

Propagation of travelling waves in a complex system modelling fire spread

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Abstract: A system of coupled partial differential equations which models a complex system of a solid fuel, endothermically pyrolysing to a combustible gas, which in turn exothermically reacts with oxygen, is studied. We use a numerical method based on the Crank-Nicholson finite difference scheme to solve the governing equations in order to investigate the dynamics of this model. It has previously been shown that there exist solutions to the model which exhibit oscillatory propagating combustion waves. In this work, we extend the original study to explore the parameter space to locate regions where both steady propagating waves (single valued wave speed) and also pulsating waves exist. Parameter space where no propagating waves are possible (extinction region) is also determined.

Keywords: *Combustion, exothermic, endothermic, pyrolysis, travelling waves, instability, bifurcation*

1 INTRODUCTION

The modelling of fire spread has been an active research topic with many applications to both urban and rural fire situations, particularly to minimise risk to humans, livestock and infrastructure (Drysdale 1999, Rahmawati et al. 2016, Peng et al. 2016). Of particular interest is the dynamics of fire spread (Grishin 1984, Weber 1991, Duane et al. 2016). The rate of spread of the combustion wave is of interest, for example in the case of wildfire spread (Watt et al. 1995, Hilton et al. 2015).

In previous work by Weber et al. (2001), a model was presented which involved the combustion of a solid fuel which underwent an endothermic gasification which then could react exothermically with oxygen, and with the heat released to propagate a combustion wave. One of the results of the work was the discovery of oscillatory propagating combustion waves for a given set of system parameters. In the present investigation, we revisit this model with the intention to determine various dynamical behavior in the parameter space.

In studying travelling wave solutions in combustion problems (see for example Volpert et al. (1994)), the existence of a stable, steady propagating combustion wave is assumed. For some systems, it is possible to obtain an asymptotic approximation to the wave speed (Weber et al. 1997). However, as with dynamical systems, it has been shown that in a system of partial differential equations (PDEs) there is the possibility of instabilities (Gubernov et al. 2003, 2004). The existence of possible bifurcation of wave speed was first noticed by Shkadinskii et al. (1971). In Weber et al. (1997), a single-step combustion model was demonstrated to have an instability as the system parameters were varied and a period-doubling route to chaos was shown. Similar analysis was carried out for more complicated models such as for a competitive exothermic-endothermic combustion problem (Sharples et al. 2012, Wee et al. 2012) and for a sequential reaction (Qian et al. 2011).

2 MATHEMATICAL MODEL

We begin with the conservation of energy and mass, for the solid and gas phases. This model is based on that presented in Weber et al. (2001) where it was assumed that the solid fuel cannot diffuse heat or mass, while the gaseous fuel can diffuse. The non-dimensionalised system can be written as

$$\frac{\partial u_s}{\partial t} = -Q_s v_s e^{-\gamma/u_s} + h_s(u_g - u_s) - \ell_s(u_s - u_a), \quad (1)$$

$$\frac{\partial v_s}{\partial t} = -\beta_s v_s e^{-\gamma/u_s}, \quad (2)$$

$$\frac{\partial u_g}{\partial t} = \frac{\partial^2 u_g}{\partial x^2} + Q_g v_g e^{-1/u_g} - h_g(u_g - u_s) - \ell_g(u_g - u_a), \quad (3)$$

$$\frac{\partial v_g}{\partial t} = \frac{1}{Le} \frac{\partial^2 v_g}{\partial x^2} - \beta_g v_g e^{-1/u_g} + \beta_s v_s e^{-\gamma/u_s}, \quad (4)$$

where u denotes temperature and v denotes fuel density, the subscripts g and s correspond to the gaseous and solid stages respectively, Q denotes heat of combustion, h is a heat transfer between the phases, ℓ is a heat loss coefficient, u_a is the ambient temperature and β are the stoichiometric coefficients of the phases. In addition there are two relative parameters, γ which is a ratio of the activation energies of the two reactions and Le is the Lewis number and is the ratio of the thermal conductivity and the mass diffusivity in the gaseous phase.

3 NUMERICAL METHOD

The system (1) – (4) was solved numerically for different stoichiometric coefficients (β_s and β_g) using the Crank-Nicholson method (see for example Press et al. (1992)) with a finite difference scheme for the spatial derivatives and adaptive finite difference for the time derivatives. The Crank-Nicholson method was chosen as it is unconditionally stable for any time step. However the time step is adjusted to satisfy error tolerances.

For each of the numerical solutions, the initial temperature and fuel profiles are

$$u_s = [1 - H(x - x^*)] u^* + u_a, \quad v_s = 1, \quad u_g = u_a \text{ and } v_g = 0,$$

where H is the Heaviside step-function, u^* and x^* are the height and width of the step function respectively. We also impose the following boundary conditions

$$\begin{aligned} \frac{\partial u_s}{\partial x} = \frac{\partial v_s}{\partial x} = \frac{\partial u_g}{\partial x} = \frac{\partial v_g}{\partial x} = 0 \text{ at } x = 0, \\ u_s = u_a \quad v_s = 1 \quad u_g = u_a \quad v_g = 0 \text{ at } x = L, \end{aligned}$$

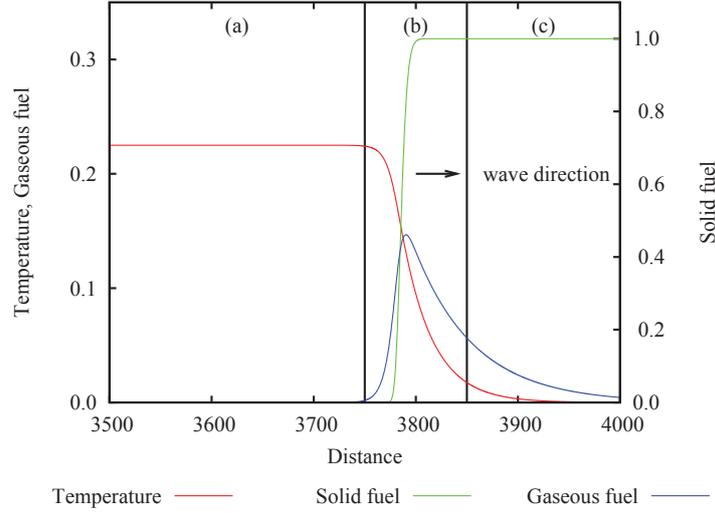


Figure 1. A schematic of the three characteristic zones in the travelling wave solution (a) the “hot” burnt zone (b) the reaction zone and (c) the “cold” unburnt zone.

where $x = 0$ and $x = L$ are the left and right boundaries respectively.

The system was then numerically integrated for a sufficiently long period so that the solution renders a steady-state solution. The progress was monitored to ensure that spatial domain was large enough to ignore boundary effects. The size of the step-function was adjusted such that there was sufficient initially energy in the system to sustain a possible combustion wave. The numerical solutions obtained by the Crank-Nicholson method were independently validated using a commercial finite element package FlexPDE (PDE Solutions Inc. 2010). The solutions obtained using both of these approaches were found to be within 1% of each other.

4 NUMERICAL RESULTS

In this work, we solve the system (1) – (4) numerically using the parameters provided in Weber et al. (2001) which were $Le = 1$, $Q_s = 0.1$, $Q_g = 1.0$, $h_s = h_g = 0.1$, $\gamma = 1.0$ and $l_s = l_g = 0$. This last condition means we are considering the adiabatic case when there is no heat loss. The two stoichiometric parameters β_g and β_s will be varied to obtain the different behavior of the travelling wave solution. In the solutions, the travelling wave is assumed to progress from the left to the right of the domain. A typical steady state solution of a combustion wave travelling with constant speed is shown in Figure 1.

There are three main regions (a) behind the travelling wave where the solid fuel has been used, pyrolysis has ceased and the temperature is in a “hot” state (b) at the wave front where the solid fuel is being pyrolysed through combustion and (c) ahead of the travelling wave where the solid fuel is unburnt and no pyrolysis has occurred. In the figure, there is also diffusion of the temperature and gaseous fuel ahead of the reaction zone.

It should be noted that as there is a coupling term between the solid and gaseous temperatures, the two temperatures quickly evolve to the same system temperature. By using this assumption, it is possible to take a linear combination of the system equations (1) – (4) to derive the following result for the temperature of the “hot” state at the left boundary

$$u = u_g = u_s = \frac{h_s h_g}{h_s + h_g} \left(\frac{Q_g}{h_g \beta_g} - \frac{Q_s}{h_s \beta_s} \right). \quad (5)$$

4.1 Steady wave speed

For the stoichiometric parameter $\beta_g = \beta_s = 2$, we have found solutions which evolve to a steady wave speed, as shown in Figure 2. In this (and subsequent figures), the combustion wave travels from left to right. As the system is spatially invariant, there would exist a combustion wave travelling in the opposite direction for $x < 0$. From the figure, the solid and gaseous fuel profiles are a constant shape over time. The temperature

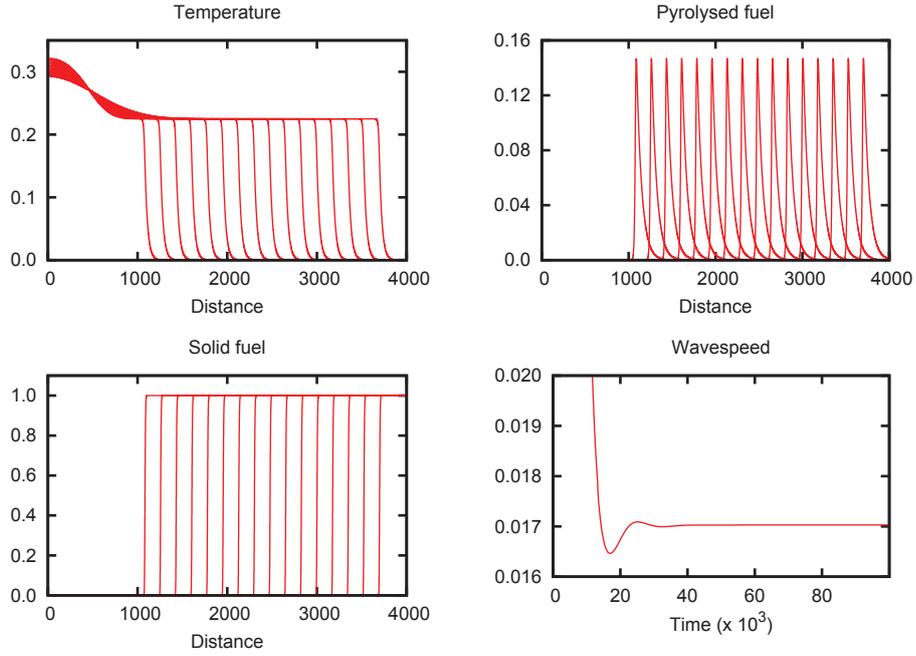


Figure 2. Travelling wave solution with steady wave speed ($c = 0.017$) with $\beta_g = \beta_s = 2$, and all parameter values given in the text.

profile is mostly a constant shape over time except on the left hand side (around $0 < x < 1000$). In this region, the wave front has passed and there is very little solid or gaseous fuel to drive the reaction. So effectively in this region, the evolution of the temperature profile (u) is governed by

$$\frac{\partial u}{\partial t} = \frac{\partial^2 u}{\partial x^2}.$$

This has a fundamental solution (see for example Watt et al. (1995))

$$u = \frac{A}{\sqrt{2\pi(t+B)}} e^{-x^2/4(t+t_1)} + C,$$

where A and B are fitted parameters and $C = u(0, t)$ is calculated from (5). So the temperature profile behind the wave front will decay at the reciprocal of the square root of time.

4.2 Oscillatory wave speed - pulsating waves

As the stoichiometric parameters are increased, as seen in the single-step model (Weber et al. 1997) and the results reported in Weber et al. (2001), an instability develops in the wave speed solution and a periodic solution for the wave speed is found for $\beta_g = \beta_s = 3$. The solution is shown in Figure 3. The solution looks fundamentally different when compared to the constant wave speed case (c.f. Figure 2). The main difference is that the profiles of the temperature and pyrolysed fuel are no longer a constant shape. In particular, the peak of the wave profiles for the temperature and pyrolysed fuel are oscillating. The other notable difference is the banding or surging of the solid fuel profile. This indicates pulsating behavior when the wave speed accelerates and decelerates in a periodic manner. We will refer to this solution as period-1 given the wave speed profile.

Increasing the stoichiometric parameters further, as has been found in similar work (for example Qian et al. (2011)), to $\beta_g = \beta_s = 3.3$, the period-1 solution becomes unstable and the solution evolves to one with a period-2 wave speed (shown in Figure 4). The period-1 and period-2 solutions shown in Figures 3 and 4 are similar in that they both show banding in the solid fuel and the oscillation of the peaks of the temperature and gaseous profiles. One of the significant differences are the different spatial and time scales between the two. Comparing the period-2 to the period-1 solutions, the numerical integration that was required to run to obtain

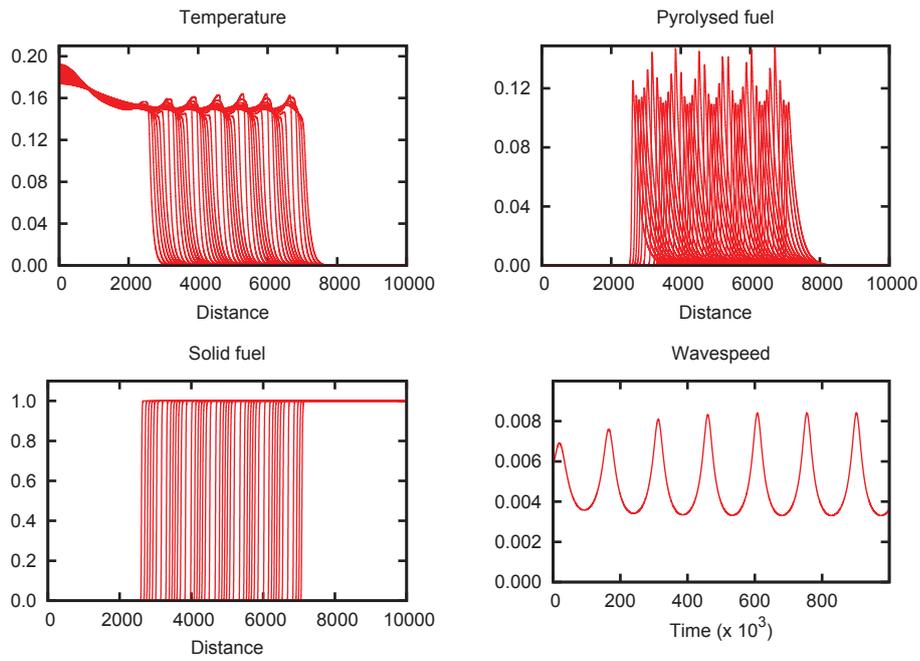


Figure 3. Travelling wave solution with oscillatory wave speed with $\beta_g = \beta_s = 3$ - period-1 solutions.

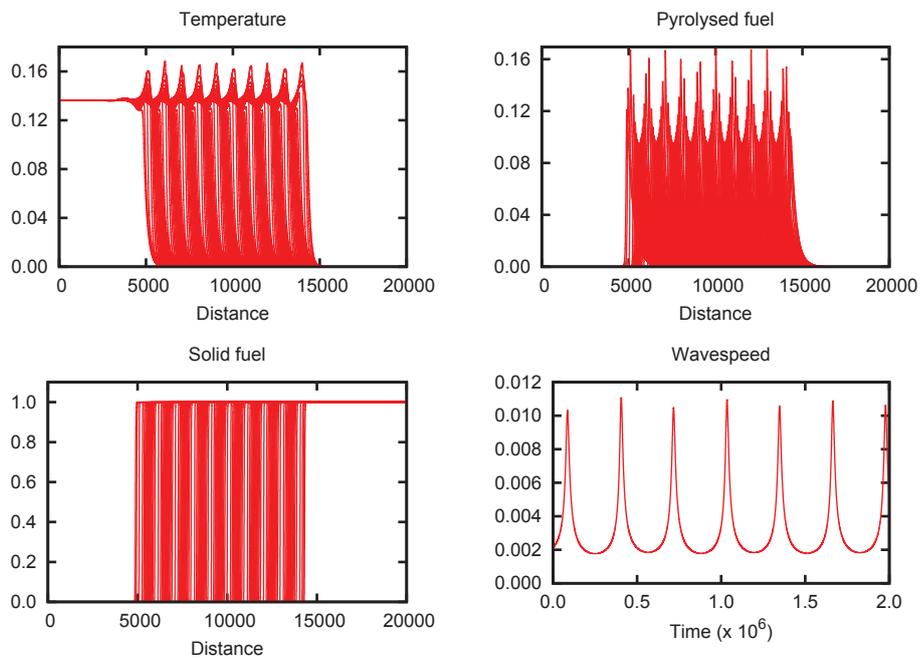


Figure 4. Travelling wave solution with oscillatory wave speed for $\beta_g = \beta_s = 3.3$ - period-2 solutions.

a steady-state solution for the former case is over a longer spatial domain and time period when compared to the period-1 solution. The average wave speed for the period-2 solution was around ten times slower.

5 EXPLORING THE PARAMETER SPACE

To get a better understanding of the system and to follow similar analysis such as that carried out in Qian et al. (2011), we explored the $\beta_s - \beta_g$ parameter space and classified the travelling wave solutions as either (a) steady wave speed (b) oscillatory wave speed (this could also include higher periodic solutions rather than the

period-1 and period-2 solutions shown in the previous section) and (c) extinction waves where no propagating combustion waves persist. For a given set of parameter values, the system (1) – (4) were numerically integrated until one of the three solution types were found.

The different types of solutions for various values of the stoichiometric parameters is shown in Figure 5. There is a Hopf curve which separates the parameter space between solutions that exhibit steady (constant speed) waves and oscillatory (pulsating) waves. This Hopf curve marks the onset of periodic behavior and was found through a manual path-following method through the parameter space. The Extinction curve describes the locus of points in the parameter space where the “hot” temperature condition found in Equation (5) was zero. In this case, the temperature diffuses away once the solid and gaseous fuels have been consumed and no combustion wave can exist. The point where the Hopf curve and the Extinction curve intersect (labelled B at $\beta_s = 0.113, \beta_g = 1.13$) is a possible Bogdanov-Takens bifurcation point, which is a critical equilibrium point (singularity) where saddle-node and Hopf bifurcations occur simultaneously. This has been observed and studied in the context of combustion waves by Gubernov et al. (2004). Whether this is in fact a Bogdanov-Takens bifurcation point will be investigated in future work.

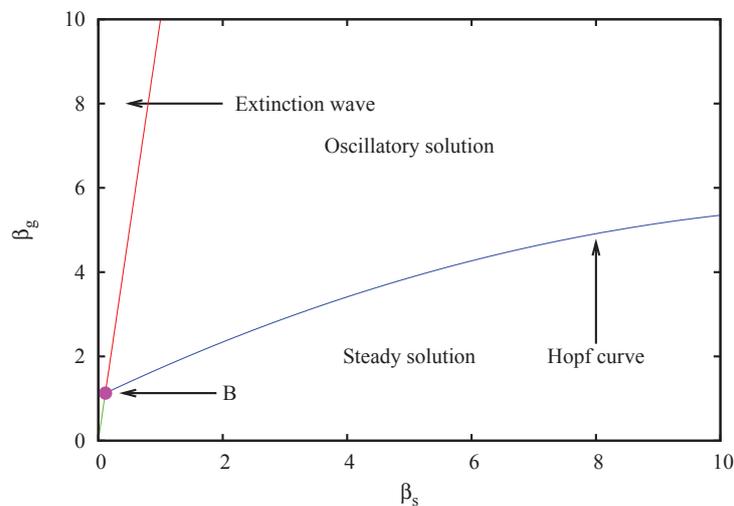


Figure 5. The $\beta_s - \beta_g$ parameter space showing the different type of waves solutions: steady speed, oscillatory speed and extinction.

6 CONCLUSIONS

In this paper, we have undertaken a preliminary investigation of a model describing the combustion scheme with two different phases, namely a solid phase and a gaseous phase. In particular, we were interested in the travelling wave behavior for different physical and chemical parameters. The model that was considered was presented in Weber et al. (2001). In this paper, we extend the original work by investigating the parameter space to locate regions where different dynamical behavior occur - steady combustion waves with a unique wave speed, pulsating waves (oscillatory wave speed) and no combustion waves. In our future work, we plan to undertake a detailed analysis of this coupled system by studying the stability of the combustion waves via the Evans function approach (see for example Gubernov et al. (2003, 2004)).

Finally, it is also important to note that using the parameter values given in Weber et al. (2001), we were not able to replicate the results presented in their paper, but instead discovered “slow” travelling wave solutions (in other words, the wave speeds were much lower than those reported in the original paper). As noted before, we have validated the numerical schemes used extensively in this work by using both the Crank-Nicholson finite difference method and the finite element package FlexPDE (PDE Solutions Inc. 2010). Hence we are confident the numerical results presented in this paper are both accurate and reliable. We intend to explore the differences between our results and those of Weber et al. (2001) further to ascertain whether there were errors in the published values given in the original paper, or there may well indeed be two families of solutions - the

“fast” and “slow” travelling wave solutions for the same parameter values. This bistability in travelling wave solutions where both “fast” and “slow” solution branches co-exist were found in the competitive exothermic-reaction scheme by Huang et al. (2016). Such a case may also exist for the system studied in this paper which could explain the differences between the results.

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